

ME 208 DYNAMICS

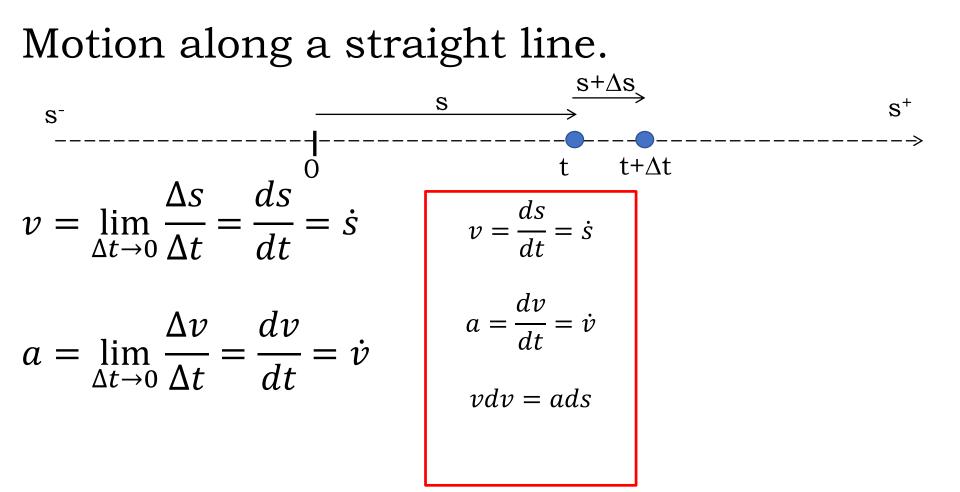
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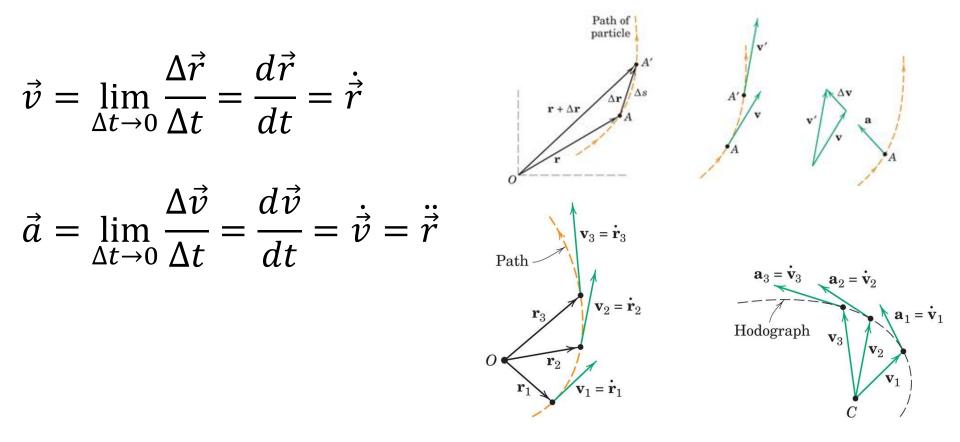


2.2 Rectilinear Motion



2.3 Plane Curvilinear Motion

Motion along a curved path in a single plane.



2.3 Plane Curvilinear Motion

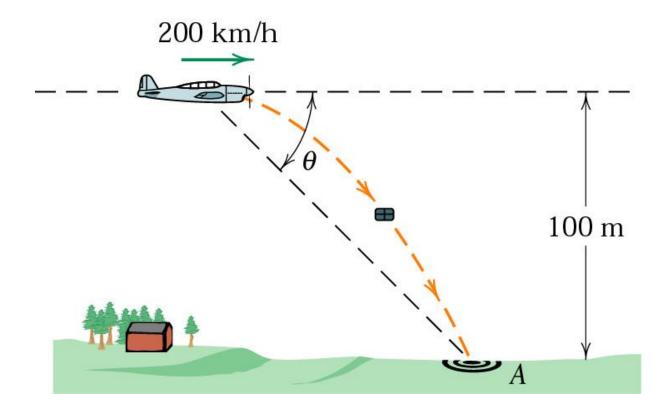
Three different coordinate systems to analyze plane curvilinear motion are:

- Rectangular (Cartesian) coordinates
- Normal-tangential (path) coordinates
- Polar coordinates

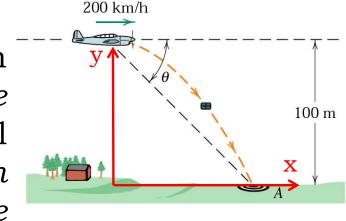
2.4 Rectangular (Cartesian) Coordinates

Rectangular coordinates is the superposition of two rectilinear motions in two mutually perpendicular directions, x and y!

2/75 (4th), 2/80 (5th), None (6th), 2/75 (7th), 2/76 (8th) The pilot of an airplane carrying a package of mail to a remote outpost wishes to release the package at the right moment to hit the recovery location A. At what angle θ with the horizontal should the pilot's line of sight to the target make the instant of release? The airplane is flying horizontally at an altitude of 100 m with a velocity of 200 km/h.



Free-fall (constant acceleration which is g) in vertical (say negative y-direction), constant speed travel in horizontal (say x-direction) when we neglect air friction on the package. $y(t) = h - \frac{1}{2}gt^2$ $x(t) = v_0 t$ $0 = 100 - \frac{1}{2}9.81t^2$ $t = \sqrt{\frac{200}{9.81}} = 4.51523641 \, s \cong 4.52 \, s$ $x = 200 \ \frac{km}{h} \frac{\frac{1000 \ m}{km}}{3600 \ s} \frac{4.52 \ s}{h} = 251 \ m$



 $r, \theta = Pol(251,100) = 270 m, 21.7^{\circ}$ Numerical accuracy versus round-off error accumulation.

Presenting Numerical Results:

Convention 1:

Always round numbers to three significant figures. Examples:

 $\sqrt{2} \approx 1.414213562 \dots \rightarrow 1.41$ 3164 m $\rightarrow 3.16 \text{ km} = 3.16 * 10^6 \text{ mm}$

Convention 2:

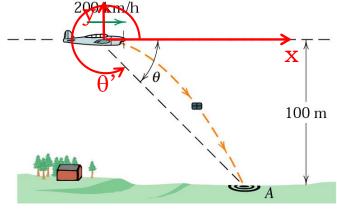
Numbers starting with a 1, use four significant figures, for other numbers use three significant figures. Examples:

 $\sqrt{2} \cong 1.414213562 \dots \rightarrow 1.412$ 3164 m \rightarrow 3.16 km = 3.16 * 10⁶ mm

Free-fall (constant acceleration which is g) in vertical (say *negative* y-direction), constant speed travel in horizontal (say x-direction) when we neglect air friction on the package. $y(t) = -\frac{1}{2}gt^2$ $x(t) = v_0 t$ $-100 = -\frac{1}{2}9.81t^2$ $t = \sqrt{\frac{200}{9.81}} = 4.51523641 \, s \cong 4.52 \, s$

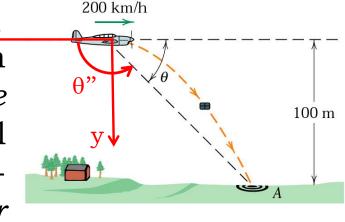
$$x = 200 \ \frac{km}{h} \frac{1000 \ \frac{m}{km}}{3600 \ \frac{s}{h}} \ 4.52 \ s = 251 \ m$$

$$r, \theta' = Pol(251, -100) = 270 m, -21.7^{\circ}$$

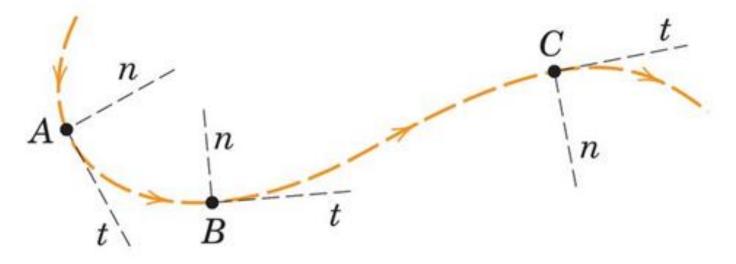


Free-fall (constant acceleration which is g) in vertical (say *positive* y-direction), constant speed travel in horizontal (say negative xdirection) when we neglect air friction on the package. $y(t) = \frac{1}{2}gt^2$ $x(t) = -v_0 t$ $100 = \frac{1}{2}9.81t^2$ $t = \sqrt{\frac{200}{9.81}} = 4.51523641 \, s \cong 4.52 \, s$ $x = -200 \ \frac{km}{h} \frac{1000 \ \frac{m}{km}}{3600 \ \frac{s}{h}} \ 4.52 \ s = -251 \ m$

 $r, \theta'' = Pol(-251,100) = 270 m, 158,3^{\circ}$



2.5 Normal & Tangential Coordinates (Path Coordinates)



The normal and tangential coordinates move with the path, t along the direction of motion, tangent to the path, n normal to t towards center of curvature of the path.

2.5 Normal & Tangential Coordinates (Path Coordinates)

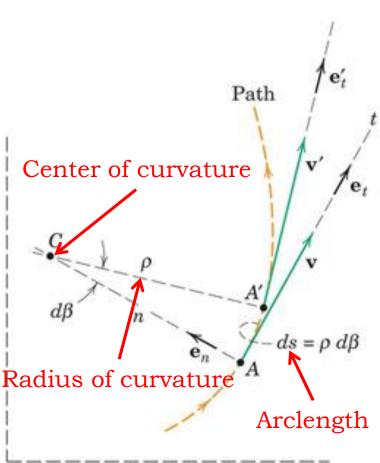
$$ds = \rho d\beta$$

$$v = \frac{ds}{dt} = \rho \frac{d\beta}{dt} + \frac{d\rho}{dt} d\beta = \rho \frac{d\beta}{dt}$$

$$\vec{v} = \rho \dot{\beta} \hat{e}_t$$

$$\vec{a} = \frac{d\vec{v}}{dt} = \frac{d}{dt} (v \hat{e}_t) = \dot{v} \hat{e}_t + v \dot{\hat{e}}_t$$

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2.5 Normal & Tangential Coordinates (Path Coordinates)

$$d\hat{e}_{t} = d\beta\hat{e}_{n}$$

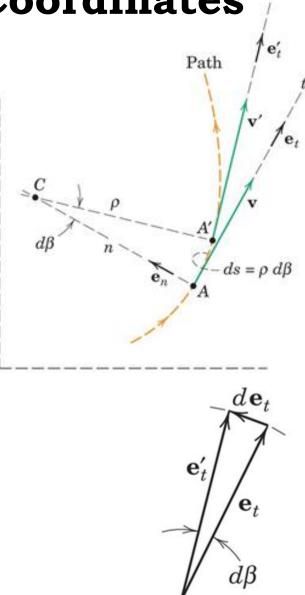
$$\dot{\bar{e}}_{t} = \frac{d\hat{e}_{t}}{dt} = \frac{d\beta}{dt}\hat{e}_{n} = \dot{\beta}\hat{e}_{n}$$

$$\vec{a} = \frac{d\vec{v}}{dt} = \frac{d}{dt}(v\hat{e}_{t}) = \dot{v}\hat{e}_{t} + v\dot{\bar{e}}_{t}$$

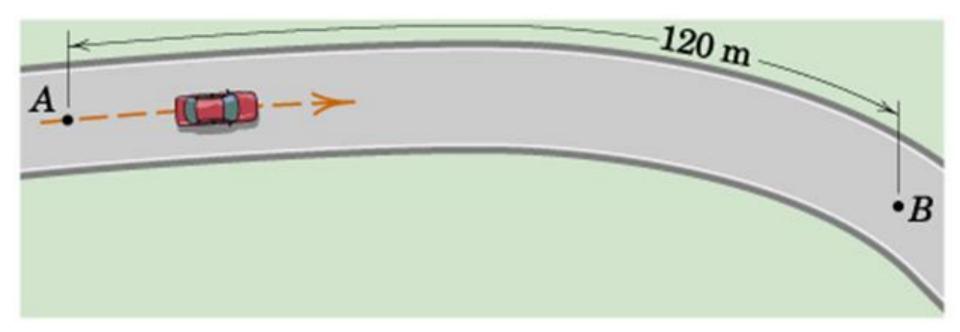
$$\vec{a} = \dot{v}\hat{e}_{t} + v\dot{\beta}\hat{e}_{n}$$

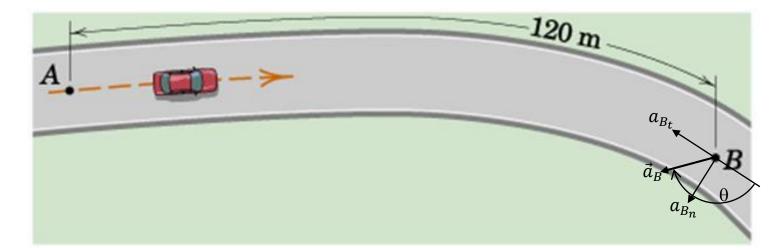
$$v = \rho\dot{\beta}, \dot{\beta} = \frac{v}{\rho}$$

$$\vec{a} = \dot{v}\hat{e}_{t} + \frac{v^{2}}{\rho}\hat{e}_{n} = a_{t}\hat{e}_{t} + a_{n}\hat{e}_{n}$$



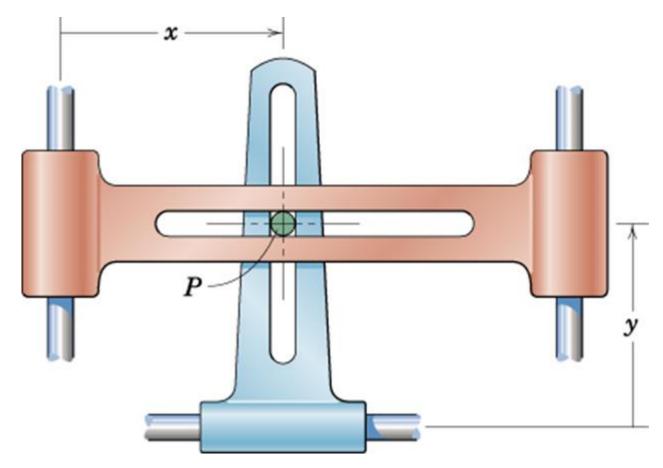
2/116 (4th), None (5th), 2/116 (6th), None (7th), None (8th) A car travels along the level curved road with a speed which is decreasing at the constant rate of 0.6 m/s each second. The speed of the car as it passes point A is 16 m/s. Calculate the magnitude of the total acceleration of the car as it passes point B which is 120 m along the road from A. The radius of curvature of the road at B is 60 m.

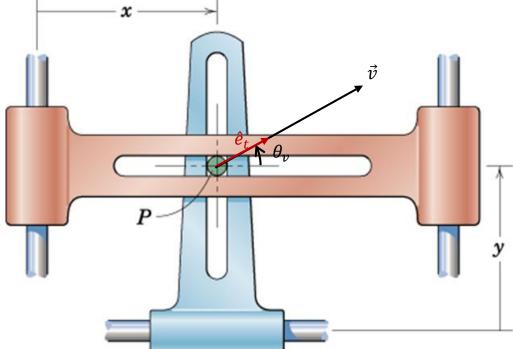




$$\begin{aligned} |\vec{a}_B| &= ?\\ \dot{v} &= a_t = -0.6 \ m/s^2\\ a_{B_n} &= \frac{v_B^2}{\rho}\\ v_B^2 &= v_A^2 + 2as = 16^2 + 2 * (-0.6)120 = 112\\ v_B &= 10.58 \ m/s\\ a_{B_n} &= \frac{v_B^2}{\rho} = \frac{10.58^2}{60} = 1.867 \ m/s^2\\ |a_B|, \theta &= Pol(a_t, a_{B_n}) = 1.960 \ m/s^2, 162.2^\circ \end{aligned}$$

2/128 (4th), None (5th), 2/132 (6th), None (7th), 2/123 (8th) During a short interval the slotted guides are designed to move according to $x = 16 - 12t + 4t^2$ and $y = 2 + 15t - 3t^2$, where x and y are in millimeters and t in seconds. At the instant when t = 2 s, determine the radius of curvature, ρ , of the path of the constrained pin P.





$$x = 16 - 12t + 4t^{2}$$

$$\dot{x} = -12 + 8t$$

$$\ddot{x} = 8$$

$$y = 2 + 15t - 3t^{2}$$

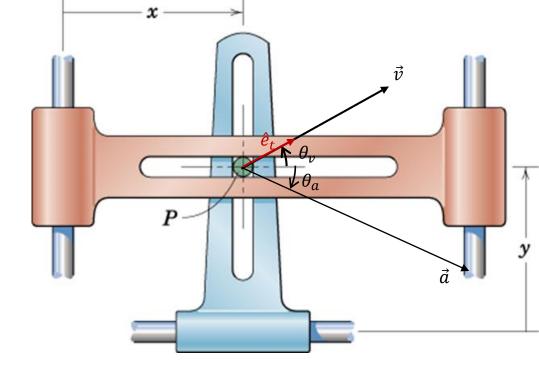
$$\dot{y} = 15 - 6t$$

$$\ddot{y} = -6$$

$$\vec{v}(t = 2s) = [(-12 + 8 * 2)\hat{i} + (15 - 6 * 2)\hat{j}] = (4\hat{i} + 3\hat{j})mm/s$$

$$\hat{e}_{t} = \frac{\vec{v}}{|\vec{v}|} = \frac{4\hat{i} + 3\hat{j}}{5}$$

$$\theta_{v} = 36.87^{\circ}$$



$$x = 16 - 12t + 4t^{2}$$

$$\dot{x} = -12 + 8t$$

$$\ddot{x} = 8$$

$$y = 2 + 15t - 3t^{2}$$

$$\dot{y} = 15 - 6t$$

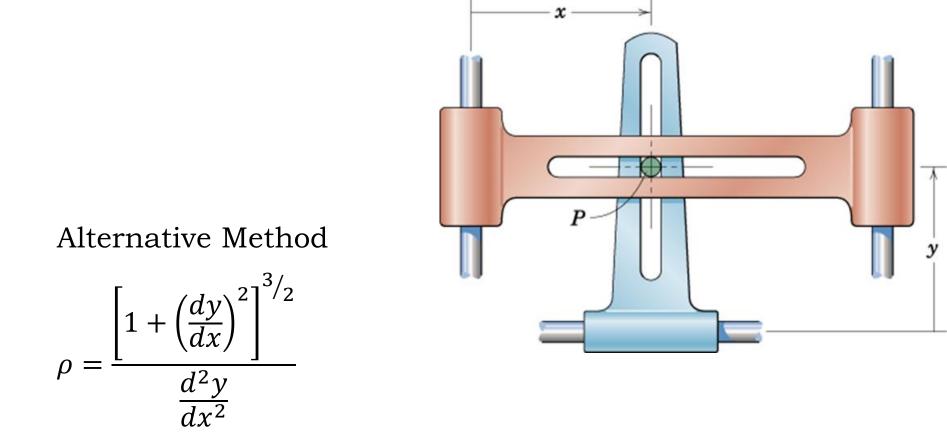
$$\ddot{y} = -6$$

$$\vec{a} = (8\hat{i} - 6\hat{j}) mm/s^{2}$$

$$\theta_{a} = -36.87^{\circ}$$

$$a_{n} = \frac{v^{2}}{\rho} = \frac{5^{2}}{\rho} = 10sin(2 * 36.87^{\circ}) = 9.60 mm/s^{2}$$

$$\rho = 2.60 mm$$



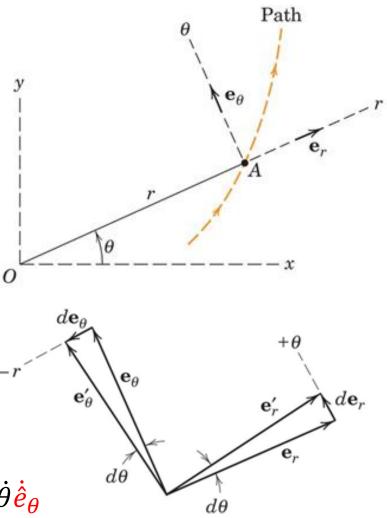
Eliminate parameter t from x and y and apply the formula to obtain the same radius of curvature or use:

$$\frac{dy}{dx} = \frac{\frac{dy}{dt}}{\frac{dx}{dt}}$$

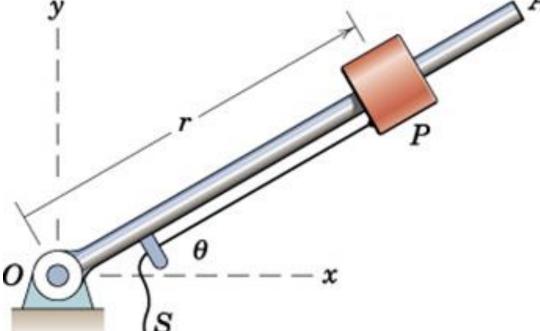
2.6 Polar Coordinates

 $\vec{r} = r\hat{e}_r$ $\dot{\vec{r}} = \vec{v} = \dot{r}\hat{e}_r + r\dot{\hat{e}}_r$

Always tangent to the path! $d\hat{e}_r = d\theta\hat{e}_{\theta}$ $\dot{\hat{e}}_r = \frac{d\hat{e}_r}{dt} = \frac{d\theta}{dt}\hat{e}_\theta = \dot{\theta}\hat{e}_\theta$ $\vec{v} = \dot{\vec{r}} = \dot{r}\hat{e}_r + r\dot{\theta}\hat{e}_A$ $\vec{a} = \dot{\vec{v}} = \ddot{\vec{r}} = \ddot{r}\hat{e}_r + \dot{r}\dot{\hat{e}}_r + \dot{r}\dot{\theta}\hat{e}_\theta + r\ddot{\theta}\hat{e}_\theta + r\dot{\theta}\dot{\hat{e}}_\theta$ $d\hat{e}_{\theta} = -d\theta\hat{e}_{r}$ $\dot{\hat{e}}_{\theta} = \frac{d\hat{e}_{\theta}}{dt} = -\frac{d\theta}{dt}\hat{e}_{r} = \dot{\theta}\hat{e}_{r}$ $\vec{a} = \dot{\vec{v}} = \ddot{\vec{r}} = (\ddot{r} - r\dot{\theta}^2)\hat{e}_r + (2\dot{r}\dot{\theta} + r\ddot{\theta})\hat{e}_\theta = a_r\hat{e}_r + a_\theta\hat{e}_\theta$



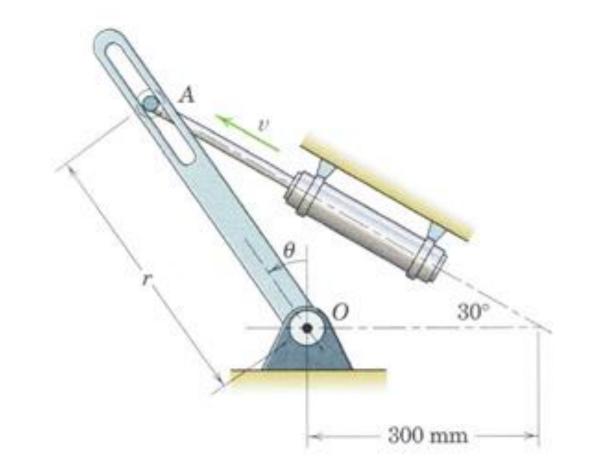
2/142 (4th), 2/144 (5th), 2/155 (6th), None (7th), None (8th) The slider P can be moved inward by means of the string S as the bar OA rotates about pivot O. The angular position of the bar is given by $\theta = 0.4 - 0.12t + 0.06t^3$ where θ is in radians and t in seconds. The position of the slider is given by $r = 0.8 - 0.1t - 0.05t^2$, where r is in meters and t in seconds. Determine and sketch the velocity and acceleration of the slider at time t = 2 s. Find the angles α and β which **v** and **a** make with positive xaxis.



$$\begin{split} \vec{v} &= \dot{\vec{r}} = \dot{r} \dot{\hat{e}}_r + r \dot{\theta} \dot{\hat{e}}_{\theta} & y & v_{\theta} \\ \vec{a} &= \dot{\vec{v}} = \ddot{\vec{r}} = (\ddot{r} - r \dot{\theta}^2) \dot{\hat{e}}_r + (2\dot{r} \dot{\theta} + r \ddot{\theta}) \dot{\hat{e}}_{\theta} = a_r \dot{\hat{e}}_r + a_{\theta} \dot{\hat{e}}_{\theta} & v_{\theta} \\ \vec{\theta}(t) &= 0.4 - 0.12t + 0.06t^3 & \dot{\theta}(t) = -0.12 + 0.18t^2 & \dot{\theta}(t) = 0.36t \\ \vec{\theta}(t) &= 0.36t & \theta(t = 2 s) = 0.640 \ rad \equiv 36.7^{\circ} & \dot{\theta}(t = 2 s) = 0.60 \ rad/s & \dot{\theta}(t = 2 s) = 0.72 \ rad/s^2 & r(t) = 0.8 - 0.1t - 0.05t^2 & \dot{r}(t) = -0.1 & r(t = 2 s) = 0.4 \ m & \dot{r}(t = 2 s) = -0.3 \ m/s & \dot{r}(t = 2 s) = -0.3 \ m/s & \dot{r}(t = 2 s) = -0.3 \ m/s^2 & \dot{v}(t = 2 s) = -0.3 \ m/s^2 & \dot{v}(t = 2 s) = -0.3 \ e_r + 0.4 * 0.6 \hat{e}_{\theta} = (-0.3 \hat{e}_r + 0.24 \hat{e}_{\theta}) \ m/s & v_{\theta}v = Pol(-0.3,0.24) = 0.384 \ m/s, 141.3^{\circ} & \alpha = \theta + \theta_v = 178^{\circ} & \dot{a} = (-0.1 - 0.4 * 0.6^2) \hat{e}_r + (2 * -0.3 * 0.6 + 0.4 * 0.72) \hat{e}_{\theta} \\ &= (-0.224 \hat{e}_r - 0.072 \hat{e}_{\theta}) \ m/s^2 & a_{\theta}a = Pol(-0.224, -0.072) = 0.235 \ m/s^2, -162.2^{\circ} & \beta = \theta + \theta_a = -125.5^{\circ} \ or \ 234.5^{\circ} & \dot{\theta} \\ \end{aligned}$$

2/142 (4th)

The hydraulic cylinder gives pin A a constant velocity v = 2 m/s along its axis for an interval of motion and, in turn, causes the slotted arm to rotate about O. Determine the values of \dot{r} , \ddot{r} and $\ddot{\theta}$ for the instant when $\theta = 30^{\circ}$.



$$r = 0.3 m$$

$$v_r = \dot{r} = v \cos(30^\circ) = 1.732 m/s$$

$$v_{\theta} = r\dot{\theta} = v \sin(30^\circ) = 1 m/s$$

$$\dot{\theta} = \frac{v_{\theta}}{r} = \frac{1}{0.3} = 3.33 rad/s$$

$$\vec{a} = \vec{0} \rightarrow a_r = 0 \text{ AND } a_{\theta} = 0$$

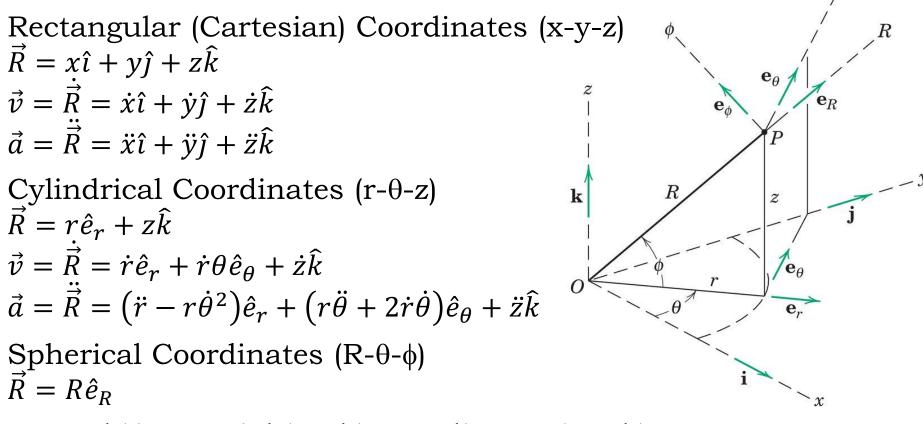
$$a_r = \ddot{r} - r\dot{\theta}^2$$

$$\ddot{r} = r\dot{\theta}^2 = 0.3 * 3.33^2 = 3.33 m/s^2$$

$$a_{\theta} = r\ddot{\theta} + 2\dot{r}\dot{\theta}$$

$$\ddot{\theta} = \frac{-2\dot{r}\dot{\theta}}{r} = \frac{-2 * 1.732 * 3.33}{0.3} = -38.5 rad/s^2$$

2.7 Space Curvilinear (3-D) Motion



Normal-Tangential (Path) Coordinates (n-t-b) $\hat{e}_b = \hat{e}_t \times \hat{e}_n$