

ME 208 DYNAMICS

Dr. Ergin TÖNÜK

Department of Mechanical Engineering Graduate Program of Biomedical Engineering tonuk@metu.edu.tr

http://tonuk.me.metu.edu.tr





 $\vec{v}_A = \vec{v}_B + \vec{v}_{rel} + (x\vec{\omega} \times \hat{\imath} + y\vec{\omega} \times \hat{\jmath})$

 $\vec{v}_A = \vec{v}_B + \vec{v}_{rel} + \vec{\omega} \times (x\hat{\imath} + y\hat{\jmath})$

$$\vec{v}_A = \vec{v}_B + \vec{v}_{rel} + \vec{\omega} \times \vec{r}_{rel}$$

 $\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$

$\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$ Interpretation of Terms

Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.



Transformation of a Time Derivative (Transport or Coriolis Theorem)

Let \vec{V} be any vector quantity $\vec{V} = V_x \hat{\imath} + V_y \hat{\jmath}$

The time derivative of this quantity in X-Y coordinates is $(\vec{A}\vec{V})$

$$\left(\frac{dV}{dt}\right)_{X-Y} = \left(\frac{\dot{V}_{x}\hat{i} + \dot{V}_{y}\hat{j}}{dt}\right)_{X-Y} + \left(\frac{V_{x}\dot{i} + V_{y}\dot{j}}{dt}\right)_{X-Y} + \vec{\omega} \times \vec{V}$$

The term $\vec{\omega} \times \vec{V}$ is the difference between the time derivative of \vec{V} in the rotating (x-y) and non-rotating (X-Y) coordinate frames.

Relative Acceleration

Relative acceleration equation may be obtained either by taking time derivative of relative velocity equation $\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$ or by applying Transport/Coriolis Theorem. $\frac{d}{dt}\vec{v}_{A} = \frac{d}{dt}\vec{v}_{B} + \frac{d}{dt}(\vec{\omega} \times \vec{r}_{rel}) + \frac{d}{dt}\vec{v}_{rel}\dot{\hat{i}} = \vec{\omega} \times \hat{i}, \, \dot{\hat{j}} = \vec{\omega} \times \hat{j}$ $\vec{a}_A = \vec{a}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{\omega} \times \frac{d}{dt} (x\hat{\imath} + y\hat{\jmath}) + \frac{d}{dt} (x\hat{\imath} + \dot{y}\hat{\jmath})$ $\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} + \vec{\omega} \times \left[(\dot{x}\hat{\imath} + \dot{y}\hat{\jmath}) + (\dot{x}\dot{\imath} + y\dot{\jmath}) + (\ddot{x}\hat{\imath} + \ddot{y}\hat{\jmath}) + (\dot{x}\dot{\imath} + \dot{y}\dot{\jmath}) \right]$ $\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$

Relative Acceleration

Relative acceleration equation may be obtained either by taking time derivative of relative velocity equation $\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$ or by applying Transport/Coriolis Theorem. $\frac{d}{dt}\vec{v}_{A} = \frac{d}{dt}\vec{v}_{B} + \frac{d}{dt}(\vec{\omega} \times \vec{r}_{rel}) + \frac{d}{dt}\vec{v}_{rel} + \frac{d}{dt}\vec{v}_{rel} + \vec{v}_{A} = \vec{a}_{B} + \vec{\omega} \times \vec{r}_{rel} + \vec{\omega} \times \vec{r}_{rel} + \frac{d}{dt}\vec{v}_{rel} + \frac{d}{dt}\vec{v}_{rel} = \vec{\omega} \times \vec{v} + (\vec{v}_{rel})$ $\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} + \vec{\omega} \times (\vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}) + (\vec{\omega} \times \vec{v}_{rel} + \dot{\vec{v}}_{rel})$ $\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$ $\mathbf{r} = \mathbf{r}_{A/B}$

Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.

$$\vec{a}_{A} = \vec{a}_{B} + \vec{\alpha} \times \vec{r}_{rel} - \omega^{2} \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$

$$\psi \qquad \psi$$

$$\vec{a}_{A} = \vec{a}_{B} + \vec{a}_{P/B} + \vec{a}_{A/P}$$

$$y \qquad v_{A/B} \qquad v_{rel} = v_{A/P}$$

Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.

$$\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$



Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.

$$\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$











https://www.youtube.com/watch?v=Wda7azMvabE

BBCF

https://www.youtube.com/watch?v=Wda7azMvabE

$\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$ Interpretation of Terms

Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.



Assume the moving coordinate system x-y be formed of a plate with a slot in it where particle A can move in relative to the moving coordinate system.

$$\vec{a}_A = \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$



Sample Problem 5/16

At the instant represented, the disk with radial slot is rotating about O with a counterclockwise angular velocity of 4 rad/s which is decreasing at the rate of 10 rad/s². The motion of slider A is separately controlled, and at this instant, r is 150 mm, increasing with $\dot{r} =$ 125 mm/s and $\ddot{r} = 2025$ mm/s². Determine the absolute velocity and acceleration of A for this position.



In the solution presented by the textbook the formula is directly applied. However one may think point P fixed on the disk and instantly coincident with the slider A and obtain absolute velocity and acceleration of point A. $\vec{v}_A = \vec{v}_P + \vec{v}_{A/P}$

Since point P fixed on the rotating disk it makes fixed axis rotation about O,

$$\vec{v}_{P} = \vec{\omega} \times \vec{r}_{P} = 4\hat{k} \times 0.15\hat{i} = 0.600\hat{j} \text{ m/s}$$

$$\vec{v}_{A/P} = \dot{\vec{r}} = 0.125\hat{i} \text{ m/s}$$

$$\vec{v}_{A} = 0.6\hat{j} + 0.125\hat{i} = (0.1250\hat{i} + 0.600\hat{j}) \text{ m/s}$$

$$\vec{a}_{A} = \vec{a}_{P} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$

$$\vec{a}_{P} = \vec{\alpha} \times \vec{r}_{rel} - \omega^{2}\vec{r}_{rel} = -10\hat{k} \times 0.15\hat{i} - 4^{2}0.15\hat{i}$$

$$\vec{a}_{P} = (2.40\hat{i} - 1.500\hat{j}) \text{ m/s}^{2}$$

$$\vec{a}_{A} = 2.40\hat{i} - 1.500\hat{j} + 2 * 4\hat{k} \times 0.125\hat{i} + 2.025\hat{i}$$

$$\vec{a}_{A} = (-0.375\hat{i} - 0.500\hat{j}) \text{ m/s}^{2}$$

5/150 (4th), 5/151 (5th), None (6th), 5/159 (7th), None (8th) The disk rotates about a fixed axis through O with angular velocity $\omega = 5$ rad/s and angular acceleration $\alpha = 3$ rad/s² at the instant represented, in the directions shown. The slider A moves in the straight slot. Determine the absolute velocity and acceleration of A for the same instant, when y = 250 mm, $\dot{y} = -600$ mm/s and $\ddot{y} = 750$ mm/s².



Again one may assume point P fixed on the disk, instantly coincident with A (or may apply the formula directly).

Velocity analysis is required first since acceleration equations contain velocities too.

$$\vec{v}_A = \vec{v}_P + \vec{v}_{A/P}$$

Since point P on the disk makes fixed axis rotation about O, $\vec{v}_P = \vec{\omega} \times \vec{r}_P = 5\hat{k} \times (-0.15\hat{i} + 0.25\hat{j}) = (-1.25\hat{i} - 0.75\hat{j}) m/s$ $\vec{v}_{A/P} = \dot{y}\hat{j} = -0.6\hat{j} \ m/s$ $\vec{v}_{A} = -1.25\hat{\imath} - 0.75\hat{\jmath} - 0.6\hat{\jmath} = (-1.250\hat{\imath} - 1.350\hat{\jmath}) m/s$ $\vec{a}_A = \vec{a}_P + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$ $\vec{a}_P = \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel}$ $\vec{a}_P = -3\hat{k} \times (-0.15\hat{i} + 0.25\hat{j}) - 5^2(-0.15\hat{i} + 0.25\hat{j})$ 150 mm $\vec{a}_P = (4.50\hat{\imath} - 5.80\hat{\jmath}) m/s^2$ $\vec{a}_A = 4.5\hat{\imath} - 5.8\hat{\jmath} + 2 * 5\hat{k} \times -0.6\hat{\jmath} + 0.75\hat{\jmath}$ $\vec{a}_A = (-10.50\hat{\imath} - 5.05\hat{\jmath}) m/s^2$

5/155 (4th), 5/156 (5th), 5/165 (6th), None (7th), 5/163 (8th) Car B is rounding the curve with a constant speed of 54 km/h, and car A is approaching car B in the intersection with a constant speed of 72 km/h. Determine the velocity which car A appears to have to an observer riding and turning with car B. The x-y axes are attached to car B. Is this apparent velocity the negative of velocity that B appears to have to a nonrotating observer in car A? The distance separating two cars at the instant depicted is 40 m.



Here use of a point P fixed in the moving coordinate system and instantly coincident with A is hard to visualize and direct application of formula is straight forward.

 $v_A = 72 \ km/h \equiv 20 \ m/s$

 $v_B = 54 \ km/h \equiv 15 \ m/s$

When motion is observed through B using x-y coordinates it is **rotating** about the center of curvature of the road therefore

 $\omega = \frac{v_B}{r} = \frac{15}{100} = 0.1500 \ rad/s \ (CCW)$ $\vec{v}_A = \vec{v}_B + \vec{\omega} \times \vec{r}_{rel} + \vec{v}_{rel}$ $\vec{r}_{rel} = \vec{r}_{A/B} = -40\hat{\imath} m$ $20\hat{i} = 15\hat{j} + 0.15\hat{k} \times -40\hat{i} + \vec{v}_{rel}$ $\vec{v}_{rel} = (20.0\hat{\imath} - 9.00\hat{\jmath}) m/s$ Also consider car A at the center of curvature where $\vec{r}_{rel} = \vec{r}_{A/B} =$ $-100\hat{i}m$ and about to hit car B where $\vec{r}_{rel} = \vec{r}_{A/B} = \vec{0}$ For center of curvature: B $20\hat{i} = 15\hat{j} + 0.15\hat{k} \times -100\hat{i} + \vec{v}_{rel}$ $\vec{v}_{rel} = (20.0\hat{\imath} - 0.00\hat{\jmath}) \, m/s$ At the instant of hit: $20\hat{\imath} = 15\hat{\jmath} + 0.15\hat{k} \times \vec{0} + \vec{v}_{rel}$ 100 m $\vec{v}_{rel} = (20.0\hat{\imath} - 15.00\hat{\jmath}) \, m/s$

$$\begin{aligned} \vec{a}_A &= \vec{a}_B + \vec{\alpha} \times \vec{r}_{rel} - \omega^2 \vec{r}_{rel} + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel} \\ \vec{a}_A &= \vec{0} \\ \vec{a}_B &= \omega^2 \vec{r} = -0.15^2 * 100\hat{\imath} = -2.25\hat{\imath} \ m/s^2 \\ \vec{\alpha} &= \vec{0}, \vec{\omega} = \overline{const} \\ \vec{0} &= -2.25\hat{\imath} + \vec{0} - 0.15^2 * -40\hat{\imath} + 2 * 0.15\hat{k} \times (20.0\hat{\imath} - 9.00\hat{\imath}) + \vec{a}_{rel} \\ \vec{a}_{rel} &= (1.350\hat{\imath} - 6.00\hat{\imath}) \ m/s^2 \end{aligned}$$



 $5/162 (4^{th})$, None (5^{th}) , None (6^{th}) , None (7^{th}) , None (8^{th}) The slotted disk sector rotates with a constant counterclockwise angular velocity $\omega = 3$ rad/s. Simultaneously the slotted arm OC oscillates about line OB (fixed to the disk) so that θ changes at a constant rate of 2 rad/s except at the extremities of the oscillation during reversal of direction. Determine the total acceleration of the pin when $\theta = 30^{\circ}$ and its first rate is positive (clockwise).



One may assume a point P fixed on the disk and instantly coincident with A

$$\vec{a}_A = \vec{a}_P + 2\vec{\omega} \times \vec{v}_{rel} + \vec{a}_{rel}$$

 $\vec{r}_{rel} = (0.15tan\theta\hat{i} + 0.15\hat{j}) = (0.0866\hat{i} + 0.15\hat{j})m$
 $\vec{a}_P = \vec{a}_{disk} \times \vec{r}_{rel} - \omega_{disk}{}^2 \vec{r}_{rel}$
 $\vec{a}_P = \vec{0} \times (0.0866\hat{i} + 0.15\hat{j}) - 3^2(0.0866\hat{i} + 0.15\hat{j})$
 $\vec{a}_P = (-0.779\hat{i} - 1.350\hat{j}) m/s^2$
 $\vec{v}_{rel} = \dot{\vec{r}}_{rel} = \dot{x}\hat{i} = \frac{d}{dt}(0.15tan\theta\hat{i} + 0.15\hat{j})$
 $\vec{v}_{rel} = 0.15\dot{\theta}sec^2\theta\hat{i} = 0.15 * 2 * sec^230^\circ\hat{i} = 0.400\hat{i} m/s$
 $\vec{a}_{rel} = \dot{\vec{v}}_{rel} = \ddot{x}\hat{i} = \frac{d}{dt}(0.15\dot{\theta}sec^2\theta\hat{i}) = 2 * 0.15\dot{\theta}sec^2\theta tan\theta = 0.923\hat{i} m/s$ for $\ddot{\theta} = 0$
 $\vec{a}_A = (-0.779\hat{i} - 1.350\hat{j}) + 2(-2\hat{k}) \times 0.4\hat{i} + 0.923\hat{i} = (0.1230\hat{i} - 1.050\hat{j}) m/s^2$
Alternatively you could directly apply the formula with the same terms.



5/163 (4th), 5/168 (5th), 5/178 (6th), 5/176 (7th), 5/179 (8th) For the instant represented, link CB is rotating counterclockwise at a constant rate N = 4 rad/s, and its pin A causes a clockwise rotation of the slotted member ODE. Determine the angular velocity ω and angular acceleration α of ODE for this instant.



Assume a point P fixed on body ODE instantly coincident with A.



Assume a point P fixed on body ODE instantly coincident with A.

 $\vec{a}_P = \vec{a}_A + \vec{a}_{P/A}$ $\vec{a}_P = \vec{a}_{P_t} + \vec{a}_{P_n}$ $\vec{a}_P = \vec{\alpha}_{ODE} \times \vec{r}_{P/O} - \omega_{ODE}^2 \vec{r}_{P/O}$ $\vec{a}_P = \alpha_{ODF}\hat{k} \times 0.12\hat{i} - 4^2 * 0.12\hat{i}$ $\vec{a}_P = (-1.920\hat{i} + 0.12\alpha_{ODE}\hat{j}) m/s^2$ $\vec{a}_{A} = \vec{a}_{At} + \vec{a}_{An} = \vec{\alpha}_{CA} \times \vec{r}_{A/C} - N^{2} \vec{r}_{A/C}$ $\vec{a}_A = \vec{0} \times -0.12\hat{j} - 4^2 * -0.12\hat{j} = 1.920\hat{j} m/s^2$ $\vec{a}_{P/A} = 2\vec{\omega}_{ODE} \times \vec{v}_{P/A} + \vec{a}_{rel}$ $\vec{a}_{P/A} = 2 * -4\hat{k} \times -0.679(\cos 45^{\circ}\hat{i} + \sin 45^{\circ}\hat{j}) + a_{rel}(\cos 45^{\circ}\hat{i} + \sin 45^{\circ}\hat{j})$ $\vec{a}_{P/A} = 3.84(-\hat{\imath} + \hat{\jmath}) + a_{rel}(\cos 45^{\circ}\hat{\imath} + \sin 45^{\circ}\hat{\jmath})$ $(-1.92\hat{\imath} + 0.12\alpha_{ODE}\hat{\jmath}) = 1.92\hat{\jmath} + 3.84(-\hat{\imath} + \hat{\jmath}) + a_{rel}(\cos 45^{\circ}\hat{\imath} + \sin 45^{\circ}\hat{\jmath})$ $\hat{\iota}: -1.92 = -3.84 + a_{rel} \cos 45^{\circ}$ $a_{rel} = 2.72 \ m/s^2$ \hat{j} : $0.12\alpha_{ODE} = 1.92 + 3.84 + 2.72sin45^{\circ}$ $\alpha_{ODE} = 64.0 \ rad/s^2$